

# Single frequency force

## Physical parameters and natural frequency

$$K := 150 \cdot 10^6 \frac{N}{m} = 8.565 \times 10^5 \frac{lbf}{in}$$

stiffness and mass

$$\zeta := 0.05 \cdot 3$$

damping ratio

$$M := 300 \cdot kg = 661.387 \cdot lb$$

$$f_n := \frac{\omega_n}{2 \cdot \pi} = 112.54 \cdot Hz$$

natural frequency and

natural period of motion

$$\omega_n := \left( \frac{K}{M} \right)^{.5} = 707.107 \text{ s}^{-1}$$

$$T_n := \frac{1}{f_n} = 8.886 \times 10^{-3} \text{ s}$$

$$C := \zeta \cdot 2 \cdot (K \cdot M)^{.5} = 363.392 \cdot lbf \cdot \frac{s}{in}$$

Damping coefficient

$$F_I := 100 \cdot N$$

$$f_I := 50 \cdot Hz$$

frequency of forcing function

$$F(t, T) := F_I \cdot \sin\left(2\pi \cdot \frac{t}{T}\right)$$

$$\omega_I := f_I \cdot 2 \cdot \pi$$

$$T_{max} := (N_P - 1) \cdot \frac{1}{\Delta rate} = 0.102 \text{ s}$$

Hanning window

$$T_I := \frac{1}{f_I}$$

for graphs

$$freq_{max} := 500 \cdot Hz$$

$$T_M := T_I \cdot 10 = 0.2 \text{ s}$$

**Sampling rate = MIN = 2 x max frequency**

**Sampling rate**

$$\Delta \text{rate} := 10000 \cdot 1 \cdot \frac{1}{\text{s}} \quad \text{samples/s}$$

**Number of samples**

$$N_P := 2^{10} = 1.024 \times 10^3$$

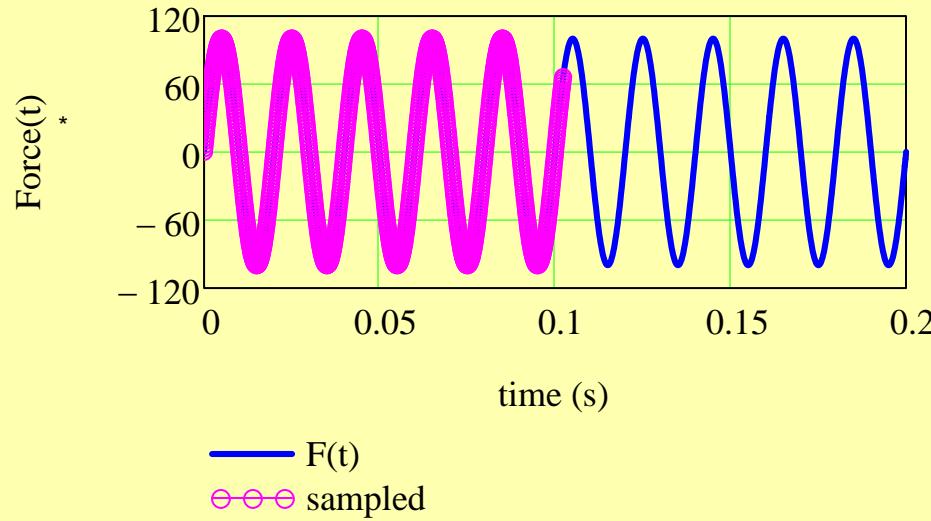
1 YES, 0: NO     $Y_{\text{window}} := 0$

Sampling must be at least twice larger than freq of forcing function

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Create function

wave form (actual and sampled)



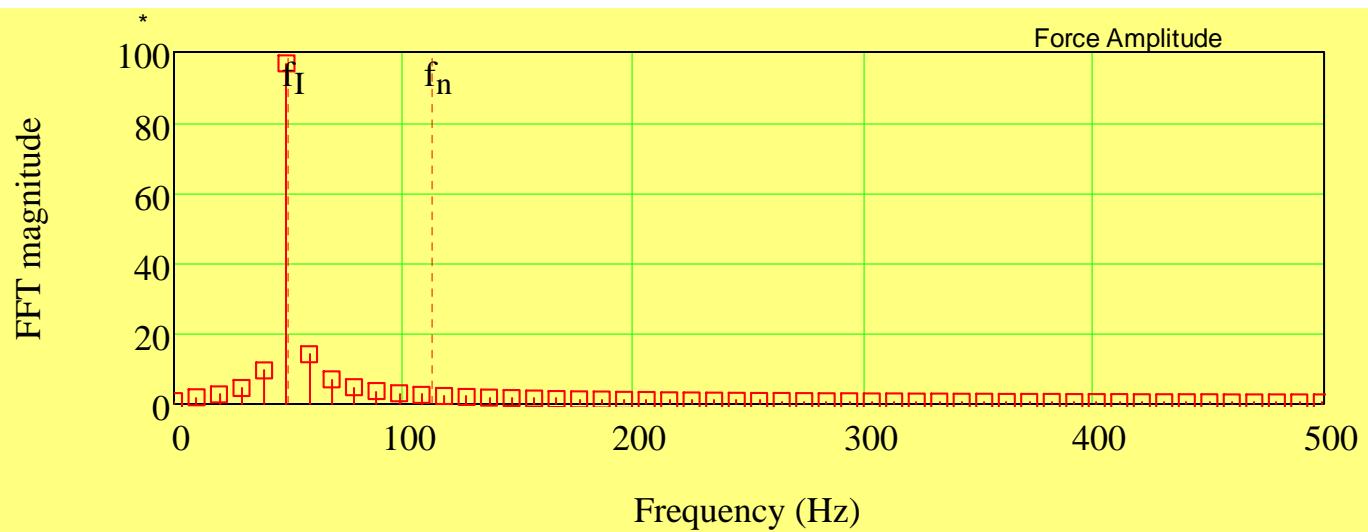
$$\frac{T_{\max}}{T_I} = 5.115 \quad \text{cycles}$$

$$\frac{\Delta t}{T_I} = 5 \times 10^{-3}$$

f=50 Hz

$$F_{I\max} := F_I$$

Create FFT



$$\Delta f = 9.785 \cdot \text{Hz}$$

$$T_{\max} = 0.102 \text{ s}$$

$$f_{\max} = 5 \times 10^3 \cdot \text{Hz}$$

$$\frac{f_{\max}}{\Delta f} = 511$$

f=50 Hz

R & I parts of FX

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## Build system response (frequency and time)

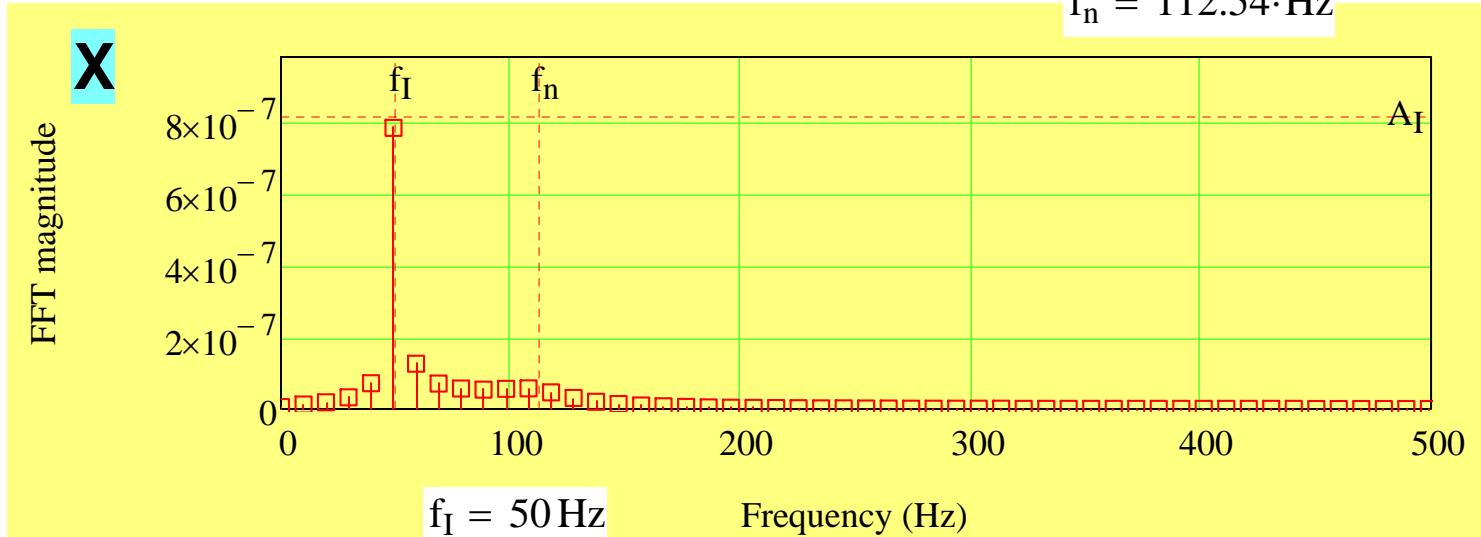
$$A_I := \frac{F_I}{\left| K - M \cdot (\omega_I)^2 + i \cdot C \cdot \omega_I \right|} = 8.194 \times 10^{-7} \text{ m}$$

exact value

$$A_{\max} := \max(A) = 7.855 \times 10^{-7} \text{ m}$$

$$\frac{A_{\max}}{A_I} = 0.959$$

$$f_n = 112.54 \cdot \text{Hz}$$



$$\Delta f = 9.785 \text{ Hz}$$

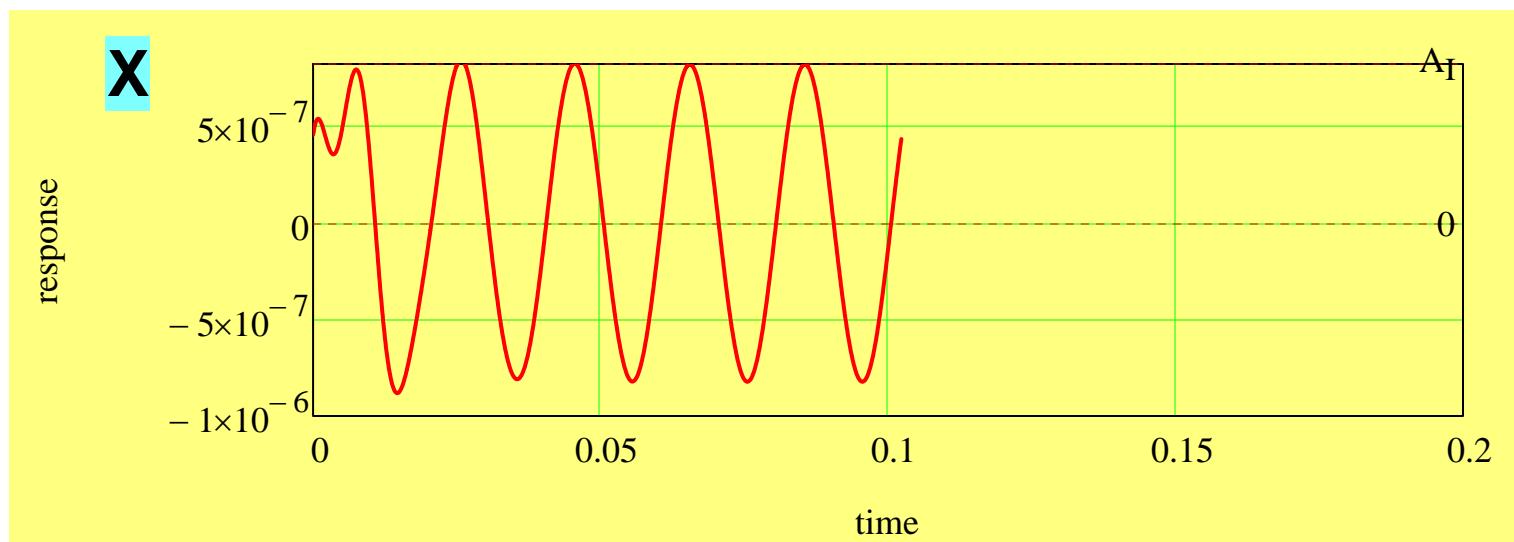
$$f_{\text{req}_9} = 78.278 \text{ Hz}$$

(b) Time response

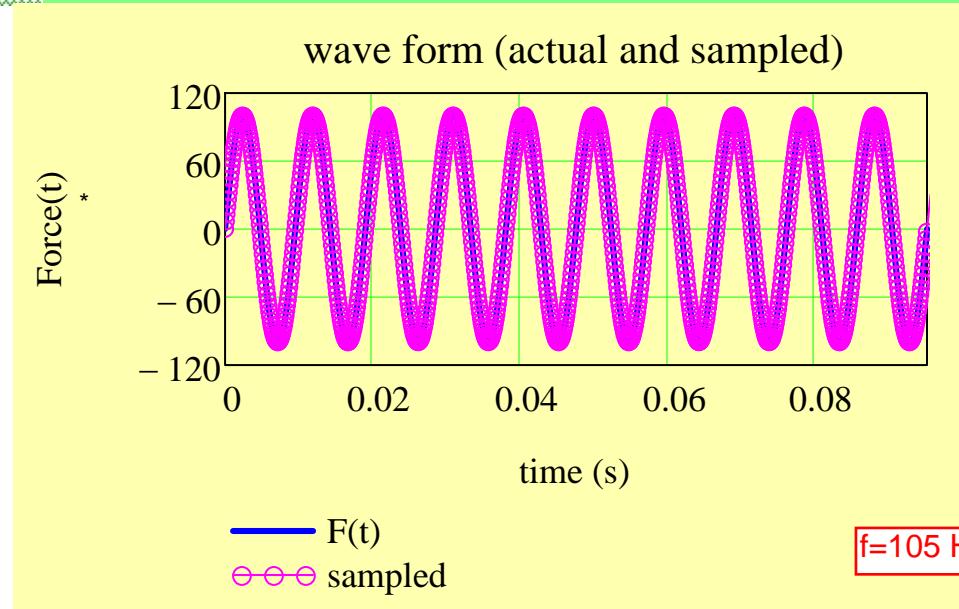
$$\max(X) = 8.39 \times 10^{-7} \text{ m}$$

$$A_I = 8.194 \times 10^{-7} \text{ m}$$

exact response



Create function

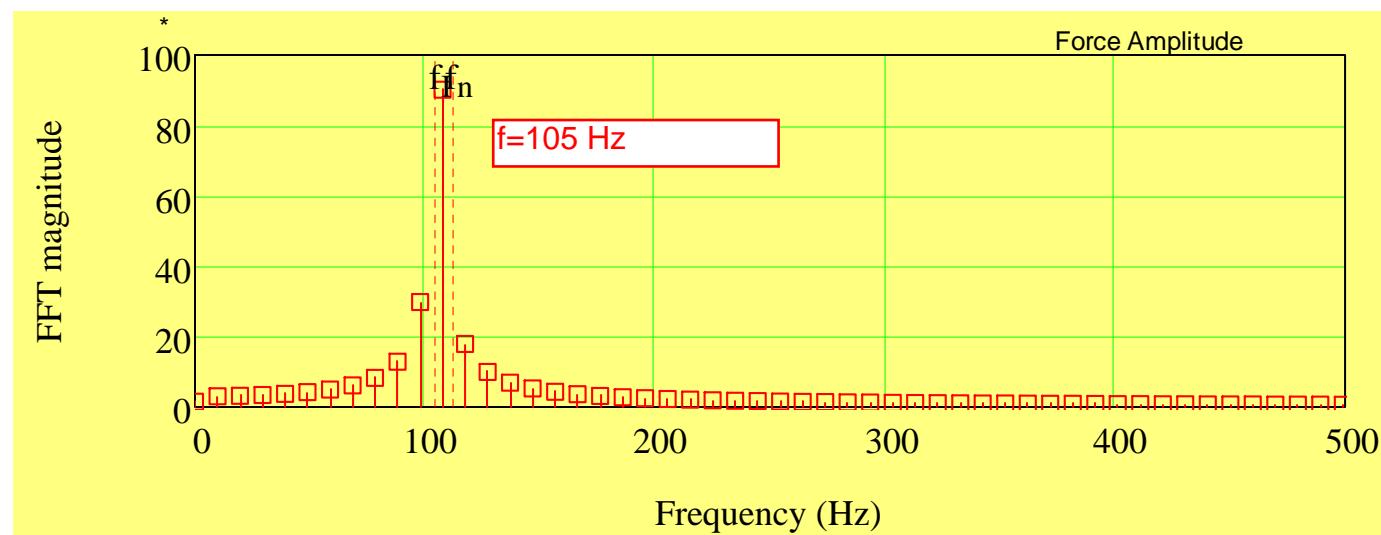


$$\frac{T_{\max}}{T_I} = 10.741 \quad \text{cycles}$$

$$\frac{\Delta t}{T_I} = 0.011$$

$$F_{I\max} := F_I$$

Create FFT



$$\Delta f = 9.785 \cdot \text{Hz}$$

$$T_{\max} = 0.102 \text{ s}$$

$$f_{\max} = 5 \times 10^3 \cdot \text{Hz}$$

$$\frac{f_{\max}}{\Delta f} = 511$$

R & I parts of FX



## Build system response (frequency and time)

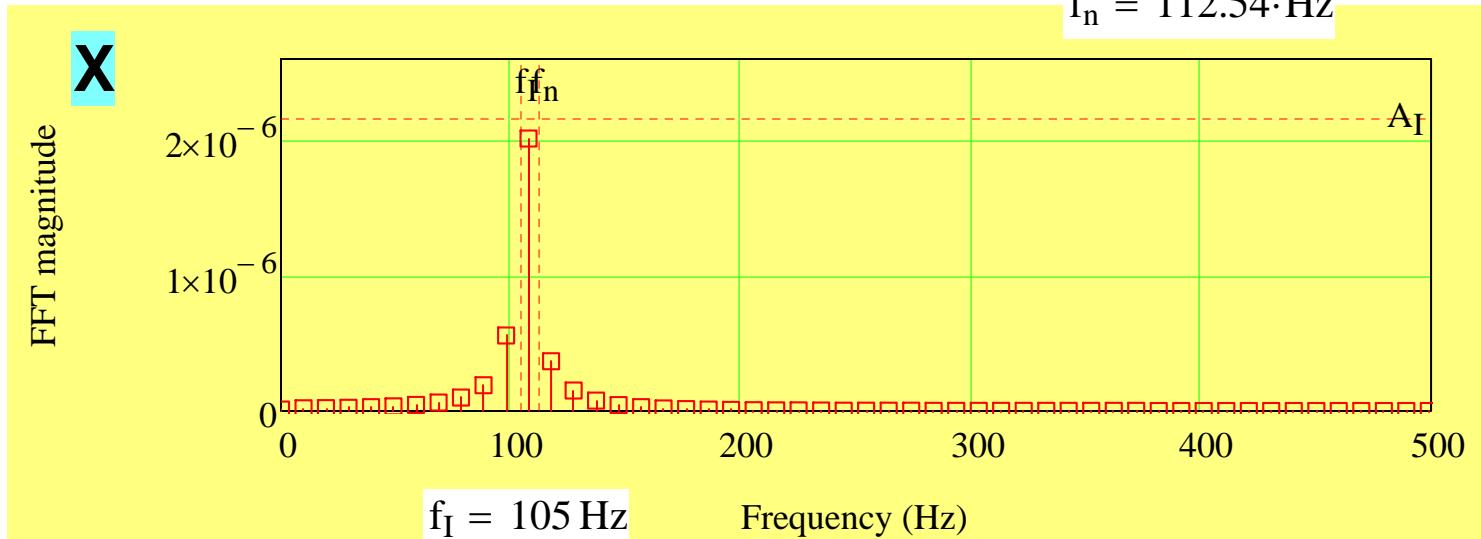
$$A_I := \frac{F_I}{\left| K - M \cdot (\omega_I)^2 + i \cdot C \cdot \omega_I \right|} = 2.162 \times 10^{-6} \text{ m}$$

exact value

$$A_{\max} := \max(A) = 2.008 \times 10^{-6} \text{ m}$$

$$\frac{A_{\max}}{A_I} = 0.929$$

$$f_n = 112.54 \cdot \text{Hz}$$



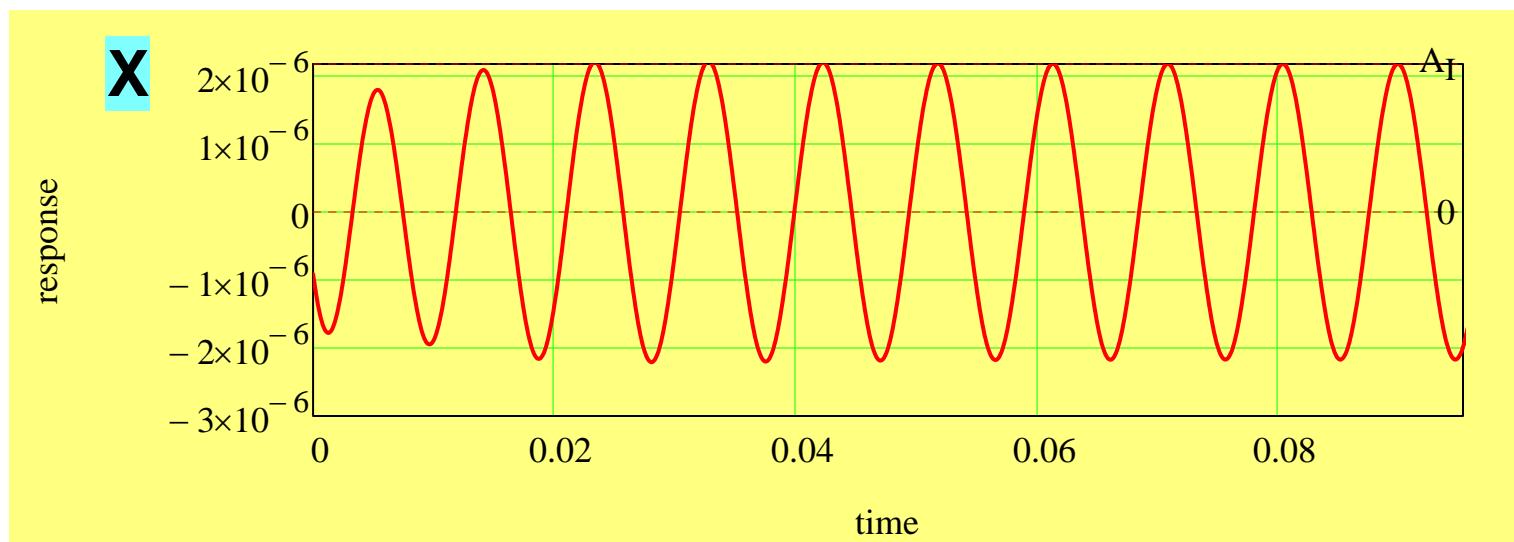
f=105 Hz

(b) Time response

$$\max(X) = 2.198 \times 10^{-6} \text{ m}$$

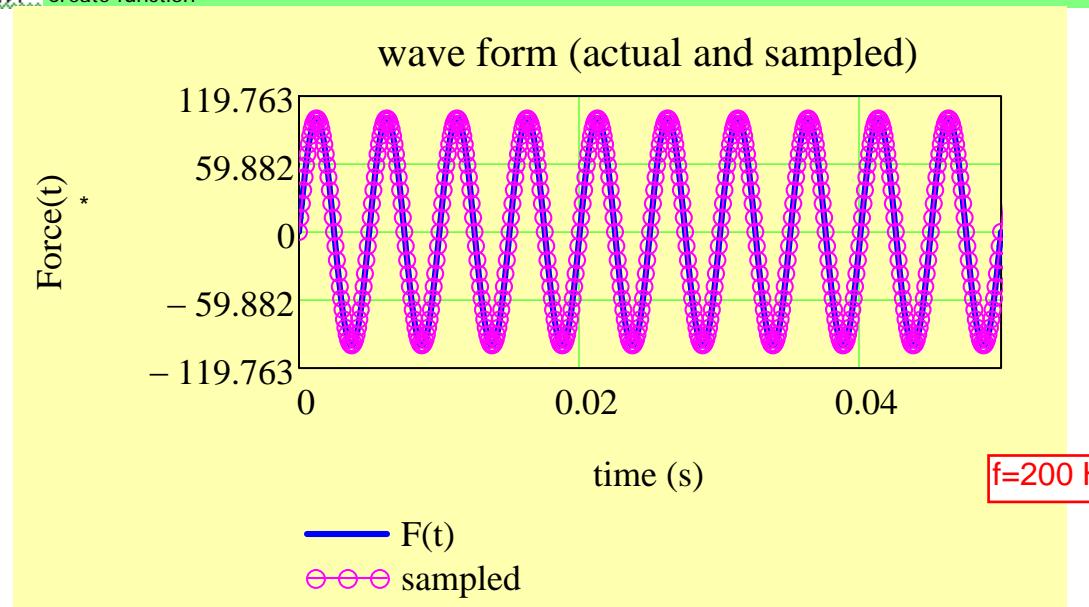
$$A_I = 2.162 \times 10^{-6} \text{ m}$$

exact response



$$f=105 \text{ Hz}$$

Create function

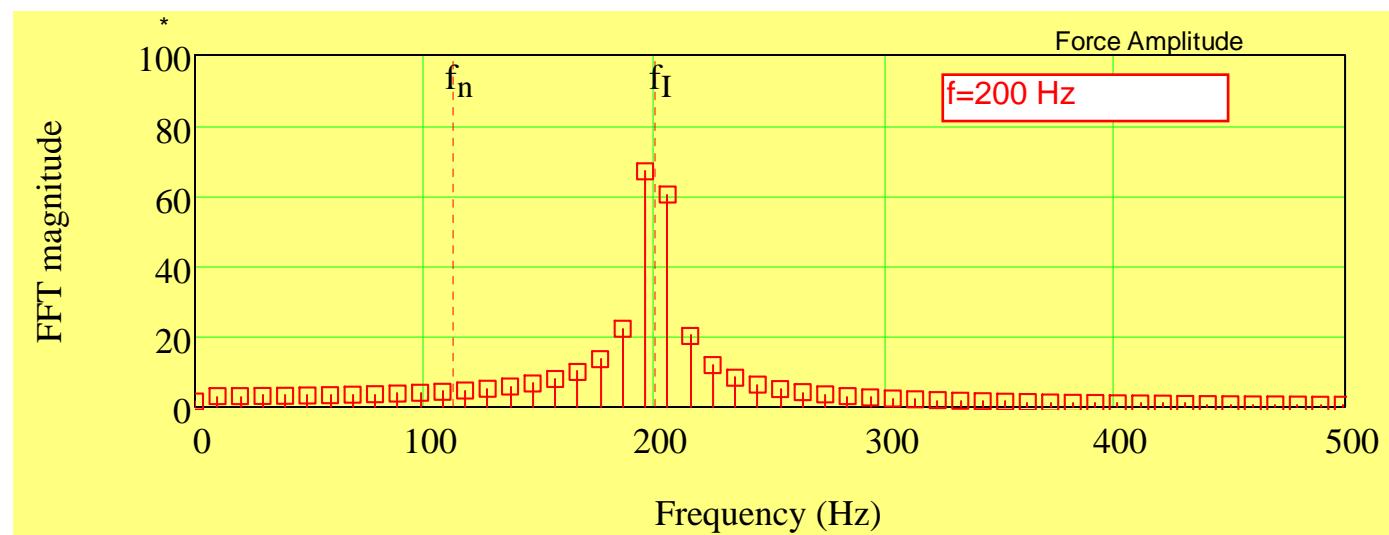


$$\frac{T_{\max}}{T_I} = 20.46 \quad \text{cycles}$$

$$\frac{\Delta t}{T_I} = 0.02$$

$$F_{I\max} := F_I$$

Create FFT



$$\Delta f = 9.785 \cdot \text{Hz}$$

$$T_{\max} = 0.102 \text{ s}$$

$$f_{\max} = 5 \times 10^3 \cdot \text{Hz}$$

$$\frac{f_{\max}}{\Delta f} = 511$$

R & I parts of FX



## Build system response (frequency and time)

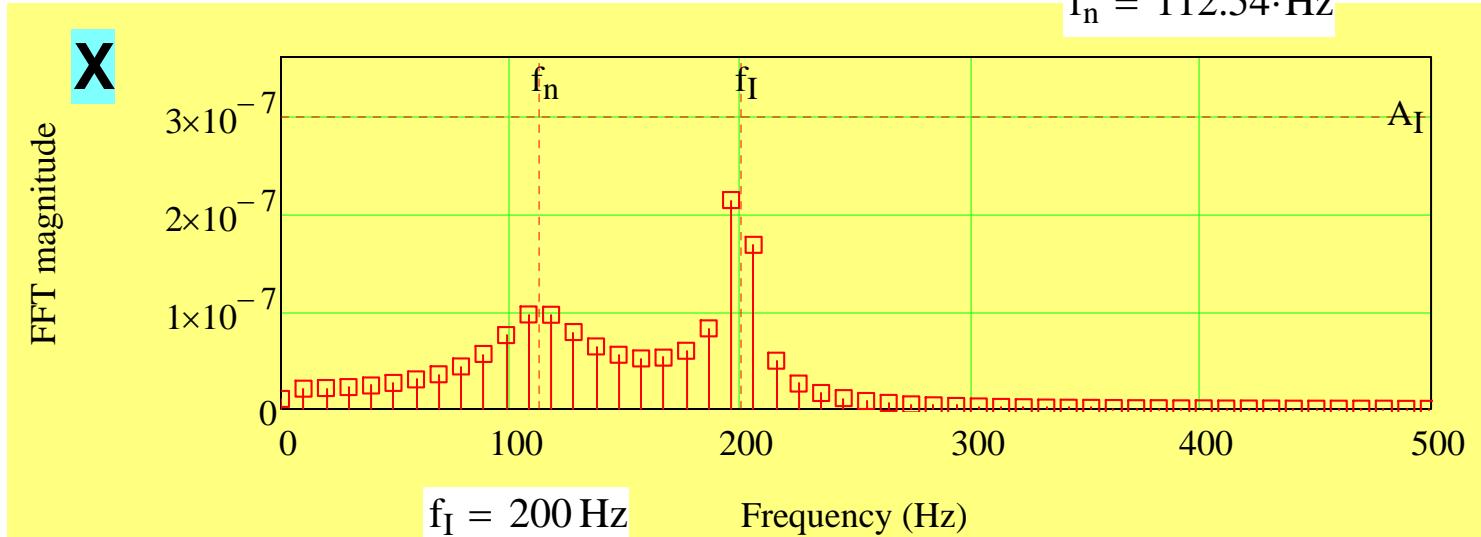
$$A_I := \frac{F_I}{\left| K - M \cdot (\omega_I)^2 + i \cdot C \cdot \omega_I \right|} = 2.999 \times 10^{-7} \text{ m}$$

exact value

$$A_{\max} := \max(A) = 2.136 \times 10^{-7} \text{ m}$$

$$\frac{A_{\max}}{A_I} = 0.712$$

$$f_n = 112.54 \cdot \text{Hz}$$

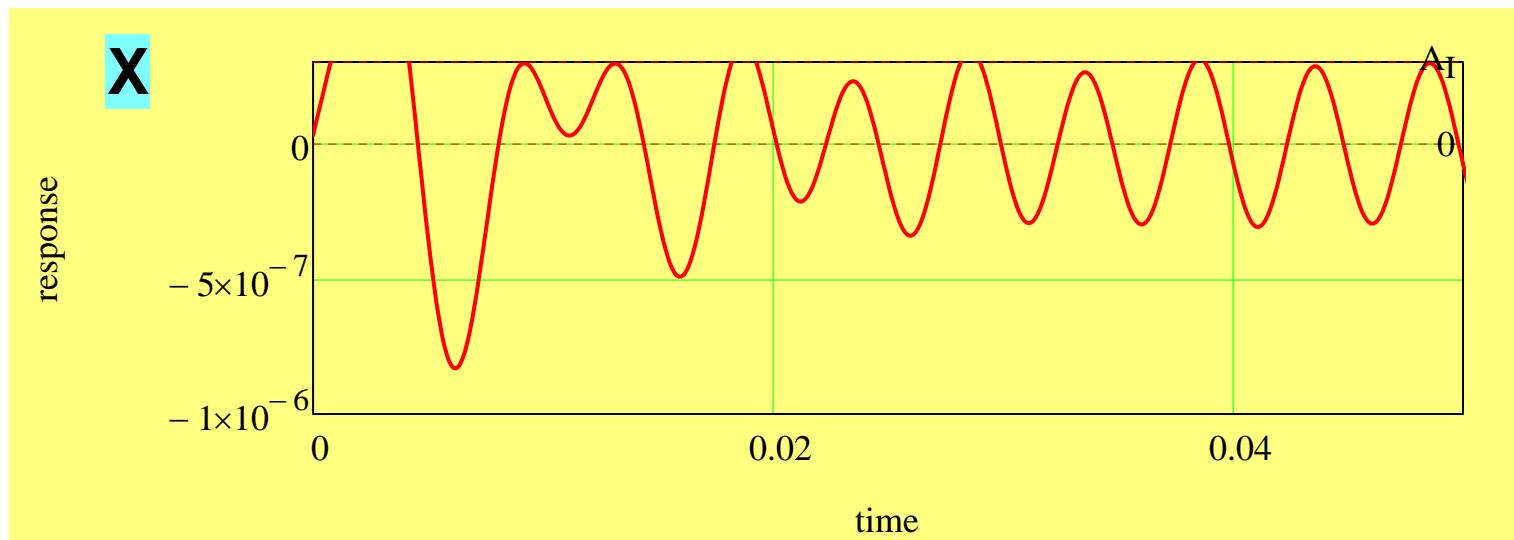


(b) Time response

$$\max(X) = 8.743 \times 10^{-7} \text{ m}$$

$$A_I = 2.999 \times 10^{-7} \text{ m}$$

exact response



f=200 Hz